

Buckingham II theorem- see also handout

Physics from dimensional analysis.

Example – pendulum.

Say we don't know the equation of motion (pre Galileo 1602), we observe – guess/intuition, etc.

Period $\tau = f(m, L, g, \theta_0 \dots)$

Now the dimensions must balance so

$$\tau \sim m^\alpha l^\beta g^\gamma \theta_0^\delta$$

ie: $[T] \equiv [M]^\alpha [L]^\beta [LT^{-2}]^\gamma [rad]^\delta$

↑
dimensionless

$[M], [L], [T]$ physical dimensions, eg: kg, m, s

Equate powers

T: $1 = -2\gamma$ $\gamma = -\frac{1}{2}$

M: $0 = \alpha$ $\alpha = 0$

L: $0 = \beta + \gamma$ $\beta = +\frac{1}{2}$

rads are dimensionless $\delta = 0$

So $\tau \sim \sqrt{\frac{l}{g}}$ independent of m and θ_0
Galileo's famous observation

Take this idea further –

"Buckingham II Theorem" 1914

Procedure

- 1) Guess the relevant quantities
- 2) System with N variables, R independent dimensions, has a solution which is function of $N-R$ groups, π_{N-R} of variables that dimensionless.
- 3) Consider the simplest π_k (Occam's razor).

Do this for the (full nonlinear) pendulum

Variable	dimension	what it is
θ_0	-	angle of release
m	[M]	mass of bob
τ	[T]	period
g	[L][T] ⁻²	grav. acceln
l	[L]	length of pendulum

$$\begin{array}{ll}
 N = 5 & \text{variables} \\
 R = 3 & \text{dimensions} \quad ([M], [L], [T])
 \end{array}$$

Therefore, $N - R = 2$ possible groups of variables $\pi_{1,2}$

Now θ_0 - dimensionless so let $\pi_1 = \theta_0$

None of the other variables has dimension M so π_2 must be independent of m .

So $\pi_2 = h(\tau, g, L)$ which is dimensionless;

$$\text{e.g.: } \pi_2 = \left(\frac{\tau^2 g}{L} \right)^\alpha \equiv \left(\frac{[T]^2 [L] [T]^{-2}}{[L]} \right)^\alpha$$

dimensionless

simplest choice $\alpha = 1$

$$\pi_2 = \frac{\tau^2 g}{L}$$

Then the pendulum's motion is given by the function

$$f(\pi_1, \pi_2) = f\left(\theta_0, \frac{\tau^2 g}{L}\right) = C$$

What is f ?

Guess that $\pi_2 = f_0(\pi_1)$ or $\tau = f_1(\theta_0) \sqrt{\frac{l}{g}}$

NB: $f_1(\theta_0)$ is universal, ie the same for any l, g .

What is $f_1(\theta_0)$?

Recall for the nonlinear undamped pendulum there is a constant of the motion

$$E = \frac{y^2}{2} - \omega^2 \cos \theta$$

where $y = \frac{d\theta}{dt}$, $\omega = \sqrt{\frac{g}{l}}$.

Now θ_0 is the angle of release of the pendulum, ie: $y = 0$ at $\theta = \theta_0$

$$E = -\omega^2 \cos \theta_0$$

then $y^2 = \left(\frac{d\theta}{dt}\right)^2 = 2E + 2\omega^2 \cos \theta = 2\omega^2 (\cos \theta - \cos \theta_0)$

$$\frac{d\theta}{dt} = \sqrt{\frac{2g}{l}} (\cos \theta - \cos \theta_0)^{1/2}$$

$$dt = \frac{d\theta}{(\cos \theta - \cos \theta_0)^{1/2}} \sqrt{\frac{l}{2g}}$$

and the τ is time for pendulum to go $\theta = 0 - \theta_0 \times 4$

so $\tau = \int_0^\tau dt = 4 \sqrt{\frac{l}{2g}} \int_0^{\theta_0} \frac{d\theta}{(\cos \theta - \cos \theta_0)^{1/2}}$

ie: $f_1(\theta_0) = 2\sqrt{2} \int_0^{\theta_0} \frac{d\theta}{(\cos \theta - \cos \theta_0)^{1/2}}$

so Π theorem \rightarrow there is some $\tau = f_1(\theta_0) \sqrt{\frac{l}{g}}$

$f_1(\theta_0)$ - universal

energy equation $\rightarrow f_1(\theta_0)$.

Π Theorem and Turbulence

Kolmogorov 1941 ideal incompressible Hydrodynamic turbulence

Variable	dimension	description
$E(k)$	$[L]^3 [T]^{-2}$	energy/unit wave number
ϵ_0	$[L]^2 [T]^{-3}$	rate of energy input
k	$[L]^{-1}$	wave number
$N = 3$	$R = 2$	so one group

$$\pi_1 = \frac{E^3 h^5}{\epsilon_0^2} = \frac{([L]^3 [T]^{-2})^3}{([L]^2 [T]^{-3})^2} \times \frac{1}{[L]^5}$$

choose $\pi_1 = \text{const}$

$$\rightarrow E = C \epsilon_0^{2/3} k^{-5/3}$$

NB: this determines the universal $-5 / 3$ exponent of ideal Kolmogorov turbulence

- does not depend on details eg viscosity.
- does not determine C .

[Actually can get C from Navier-Stokes- Kolmogorov's "4/5 law"]

Now add a variable – Magnetohydrodynamic (MHD) turbulence

Variable	Dimension	Description
$E(k)$	$[L]^3 [T]^{-2}$	energy/unit wave n_0
ϵ_0	$[L]^2 [T]^{-3}$	rate of energy $\frac{1}{\rho}$
k	$[L]^{-1}$	wave n_0
V	$[L][T]^{-1}$	characteristic speed
$\pi_1 = \frac{E^3 k^5}{\epsilon_0^2}$	$\pi_2 = \frac{V^2}{Ek}$	$= \frac{[L]^2 [T]^{-2}}{[L]^3 [T]^{-2} [L]}$

Now, let $\pi_1 \sim \pi_2^\alpha$ since we are interested in turbulence which is scaling

so $E(k) \sim k^{-(5+\alpha)/(3+\alpha)}$

α is now not determined – depends on the detailed phenomenology

↙ anomalous scaling:

may not be universal.

NB we can obtain the control parameter for turbulence in the same way:

Reynolds number for fluid turbulence

Variable	Dimension	Description
L_0	$[L]$	driving scale
η	$[L]$	dissipation scale
u	$[L][T]^{-1}$	bulk driving (flow) speed
ν	$[L]^2 [T]^{-1}$	viscosity
$P = 4 \quad R = 2 \quad M = 2$		

$$\pi_1 = \frac{UL_0}{\nu} = \frac{[L][T]^{-1}[L]}{[L]^2 [T]^{-1}} = \text{Reynolds Number}$$

$$\pi_2 = \frac{L_0}{\eta}$$

how can we relate π_1, π_2 ?

Insist on steady state so energy transfer rate same on all scales (energy rate in = energy rate out).

Mass normalised so

energy rate in $\epsilon_{inj} = \frac{U^2}{T_0} = \frac{U^3}{L_0} \quad U = \frac{L_0}{T_0}$

energy rate out – at viscous scale – obtain from Navier Stokes equation

$$\epsilon_{diss} = \frac{\nu^3}{\eta^4}$$

then steady state $\Rightarrow \epsilon_{inj} \sim \epsilon_{diss}$ so

$$\frac{U^3}{L_0} \sim \frac{\nu^3}{\eta^4}$$

or $\frac{U^3 L_0^3}{\nu^3} \sim \frac{L_0^4}{\eta^4} \quad \frac{UL_0}{\nu} \sim \left(\frac{L_0}{\eta}\right)^{4/3}$

ie steady state implies that:

$$\pi_1 = \pi_2^{4/3}$$

thus the number of excited modes or degrees of freedom (level of disorder) increases with the Reynolds number.